

PANDROL

Partners in excellence

PANDROL RAIL FASTENING SOLUTIONS FOR URBAN TRACKS: ACOUSTIC COMFORT AND ENVIRONMENTAL CARE



GET STARTED 



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Technical Manager -Pandrol Ibérica

Bonded plates Global Product Manager

Hello and Welcome

RAILWAY FORUM NAPOLI 2025

Slab Track Forum

6-7 November, Napoli, Italy

2nd International Scientific and Technical Conference
Modern Technologies in Planning, Construction, and Maintenance of
Slab Track Systems – 2025



- *Advanced degree in Industrial engineering (mechanical spec.) from the Polytechnic University of Catalonia.*
- *Master in Operations Management and Innovation Strategy from ESADE business school.*

17 years exp. on slab track

Delachaux group /

A family-owned company created in 1902, the Delachaux Group is a global player in mission-critical engineered solutions



About us...

PANDROL



DCX CHROME

① GROUPE DELACHAUX



100 years experience in the railways

40 years experience in the railways infrastructure control



WHAT WE DO / Heavy Haul



WHAT WE DO / High Speed



WHAT WE DO / Mainline



WHAT WE DO / Metro & Depot



WHAT WE DO / Light Rail & Tram



Ports & Industrial

PANDROL



Aluminothermic
Welding



Rail Fastenings
Systems



Fastening Installation
Equipment



Rail and Sleeper
Handling



Sustainable Resilient
Systems



Rail Welding
Equipment



Signalling
Equipment



Track Measurement
and Analysis



Track Safety
Equipment



Track Transportation
Equipment

1700 employees globally at the heart of more than
400 railways in over **100** countries.



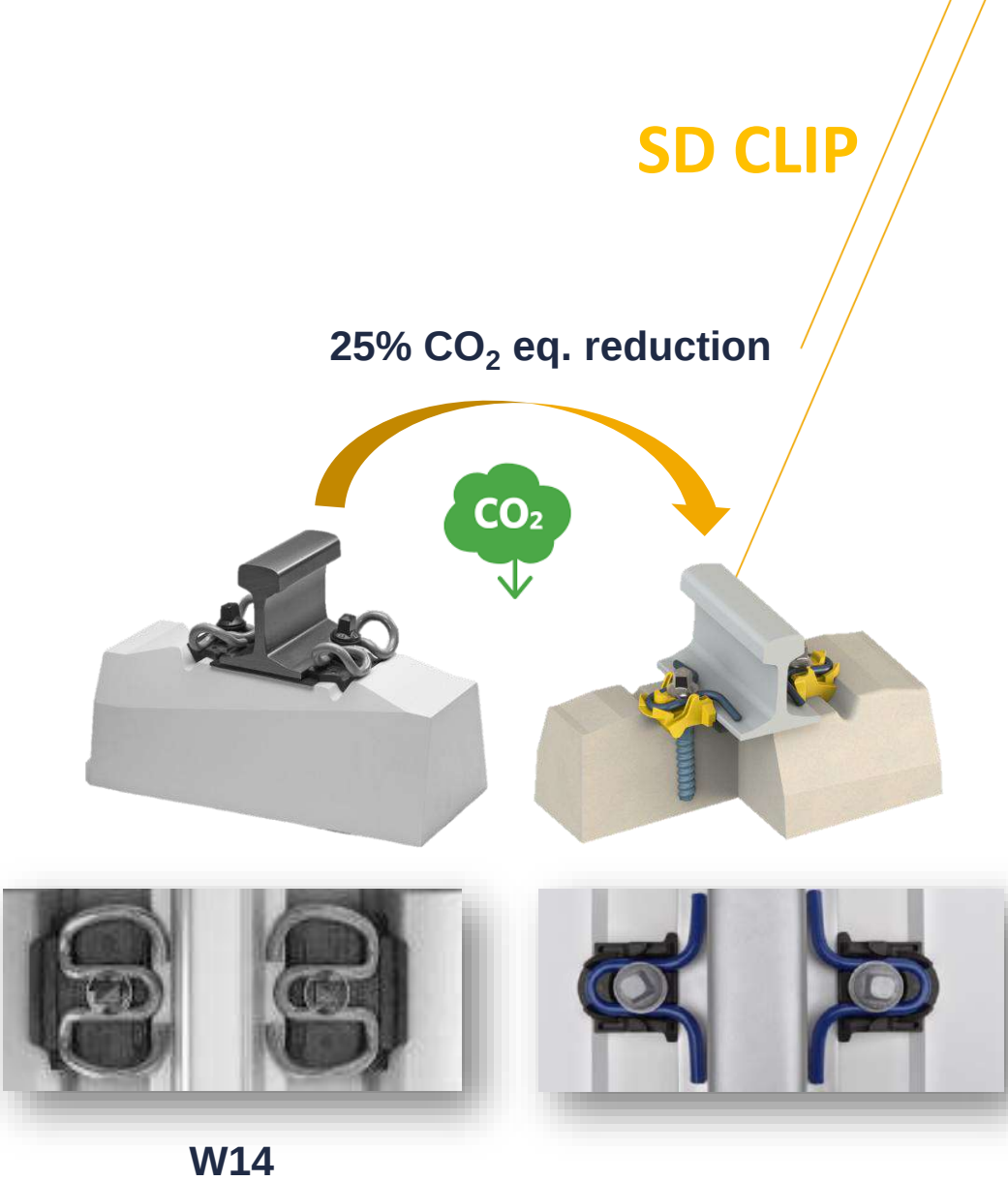
$$\text{VALUE} = \frac{\text{NEED} + \text{DIFF.}}{\text{PRICE}} \quad +\text{- SERVICE} - \text{DAMAGE (PPP)}$$



ECODESIGN

1.5-tons of raw material reduction per km of track

Component	Weight
SKL-14 Tension Clamp	30%
SD-1311 Clip	
SD-46 Insulator	25%



Pandrol SD Autodrive

Autodrive: Automated Screw-on Technology

- SD systems can be delivered with an advanced configuration that facilitates the automatic movement of the clip from the parked position to the in-service position during the screwing process.
- It is recommended to use Pandrol Autodrive installation equipment to ensure optimal functionality.



SD CLIP

SD Clip Guidance on Maintenance operations

On SD Fastenings, Clip guidance avoids dealignment or rotation during installation and maintenance operations.



SD Clip Guidance on Installation / Track construction

SD systems improve installation ratios thanks to their capacity to be pushed on when they are preassembled on the sleeper at 40 N.m.



SKL-14 Installed with Robel SKL Clipping Machine



SD Preassembled up to 40 N.m

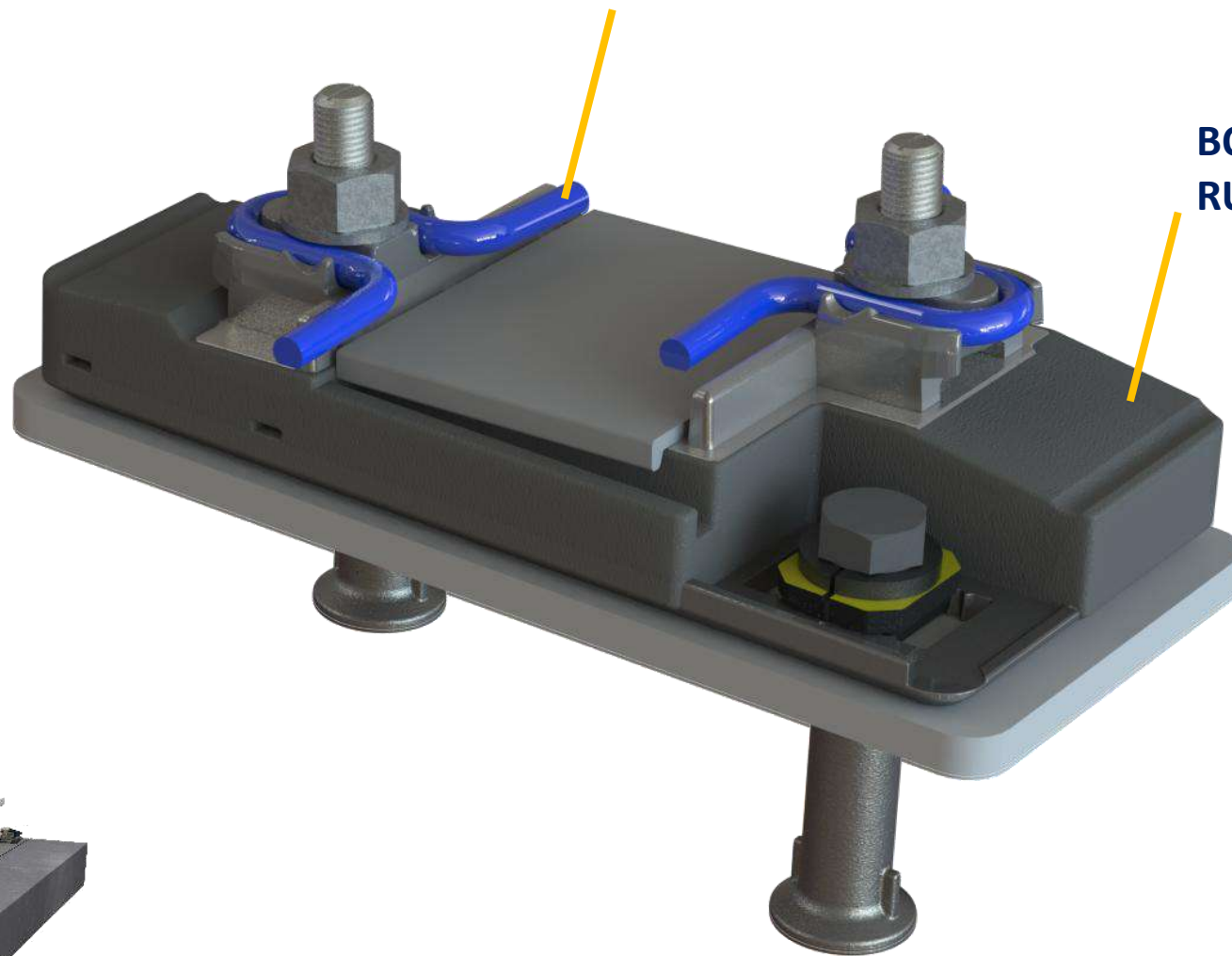
SD Clips can be also pushed with Fastclip equipment when they are preassembled up to 40 N.m



FASTENING RAIL SYSTEMS. BONDED PLATE

SD CLIP (SAFE DRIVEN)

BONDED PLATE
RUBBER-STEEL



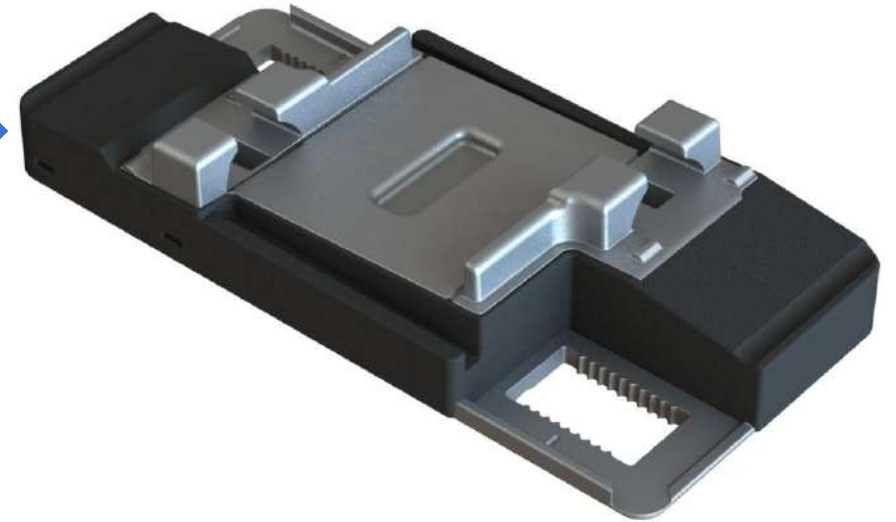
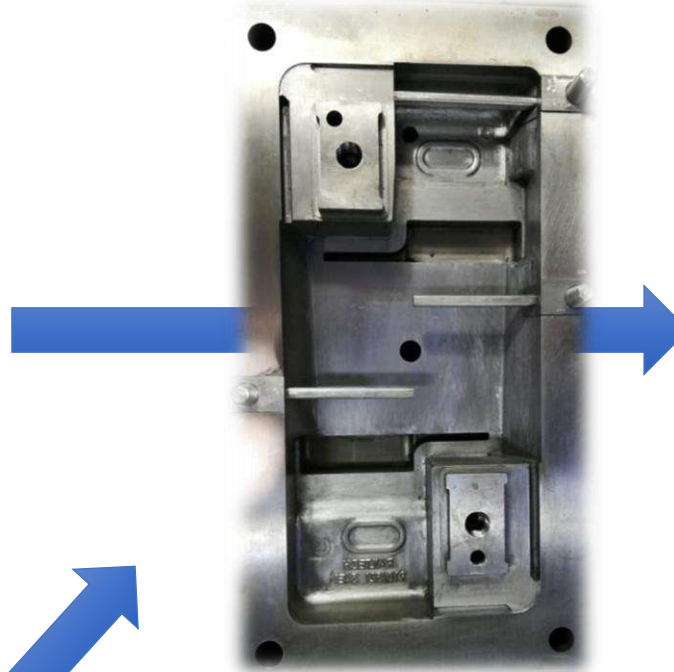
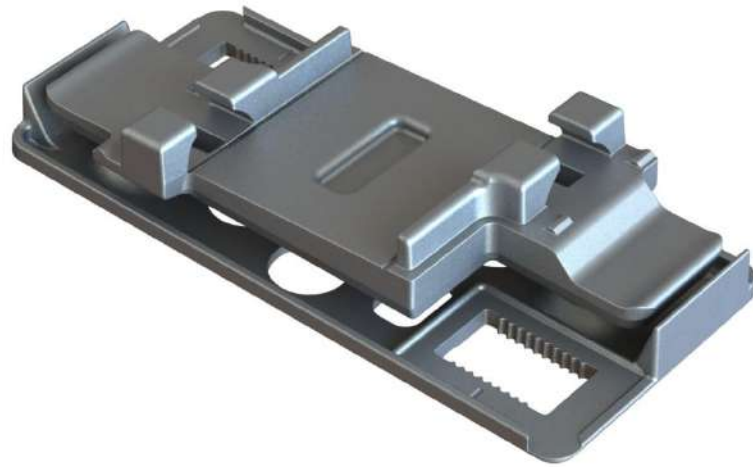
ELASTIC MAT

PANDROL

BONDED PLATES



VULCANIZATION PROCESS



MOULDING + VULCANIZATION



LIFE TIME >30years

TEST AFTER 8/14/**20** YEARS IN
SERVICE



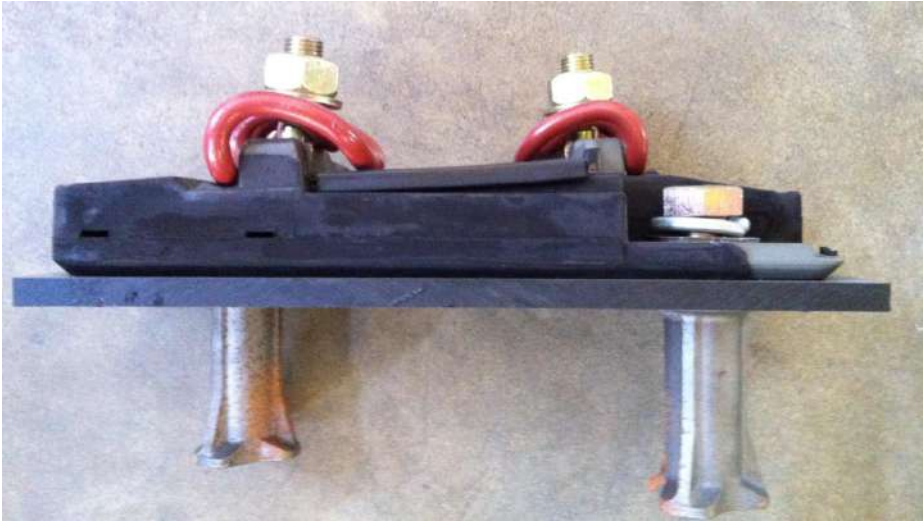
Metro de Madrid



- Stiffness recheck
- +3/6M cycles inclined RLT
- Electrical insulation
- Internal inspection



EASY TO INSTALL



PREASSEMBLED



SAFE CAST IRON INSERT



VALUE PROPOSAL

01

Low stiffness range

02

Acoustic-vibration response

03

Corrugation prevention

04

Easy to install

05

Compact

06

Guard rail version

07

Turnouts

08

Transitions

09

Durability

10

Corrosion protection

11

Electr. Isolation



Metro de Madrid







euskal trenbide sarea



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METROLINK

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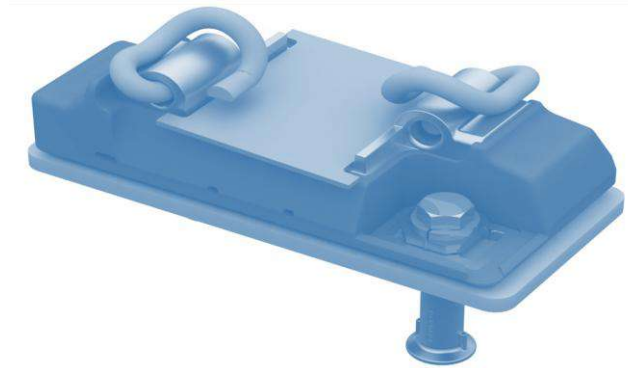
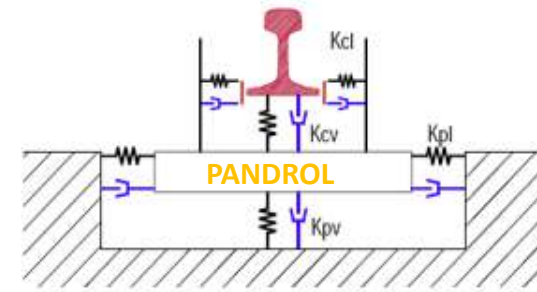








➤ STIFFNESS RANGE



k25



k17



k10

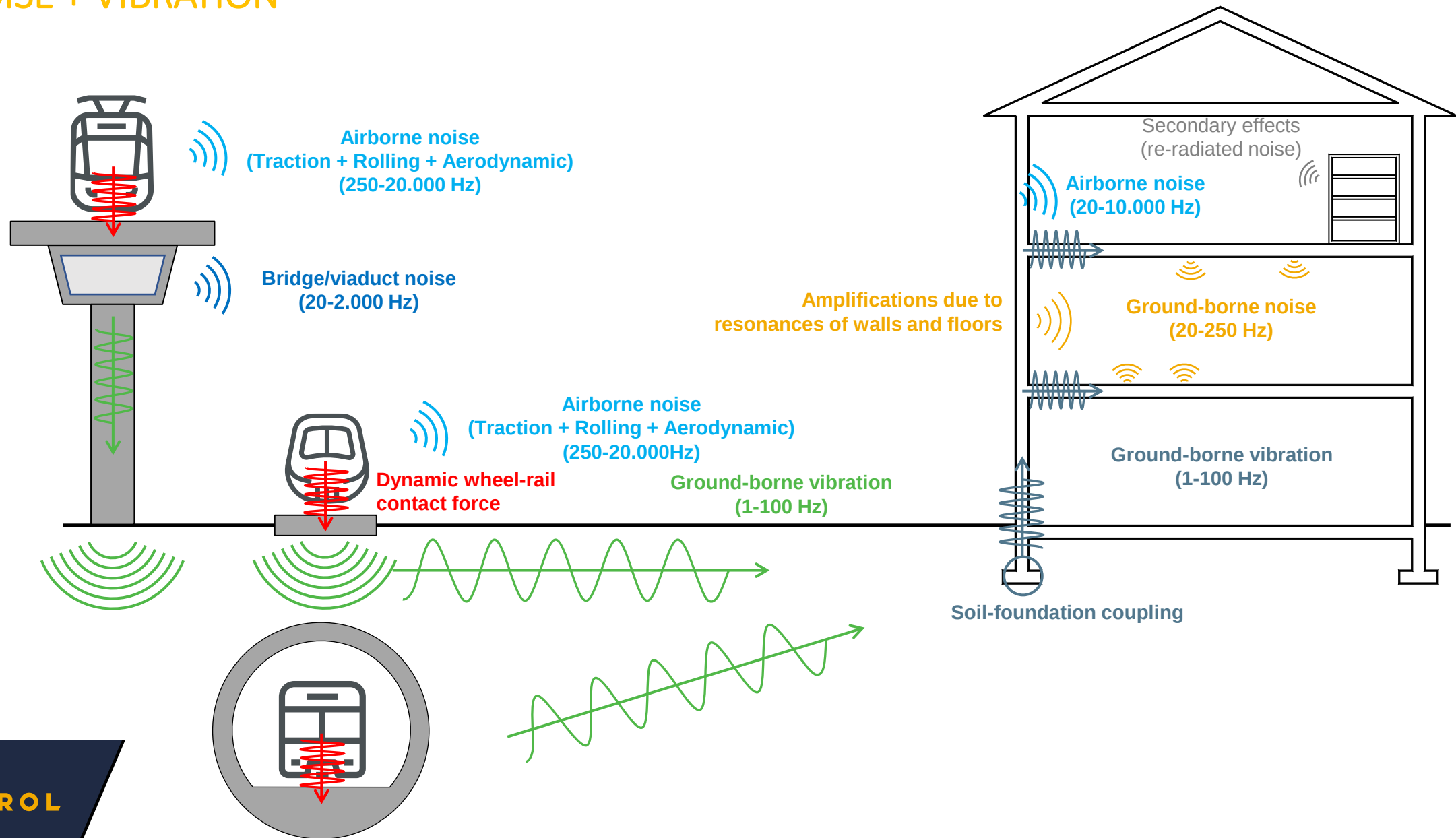


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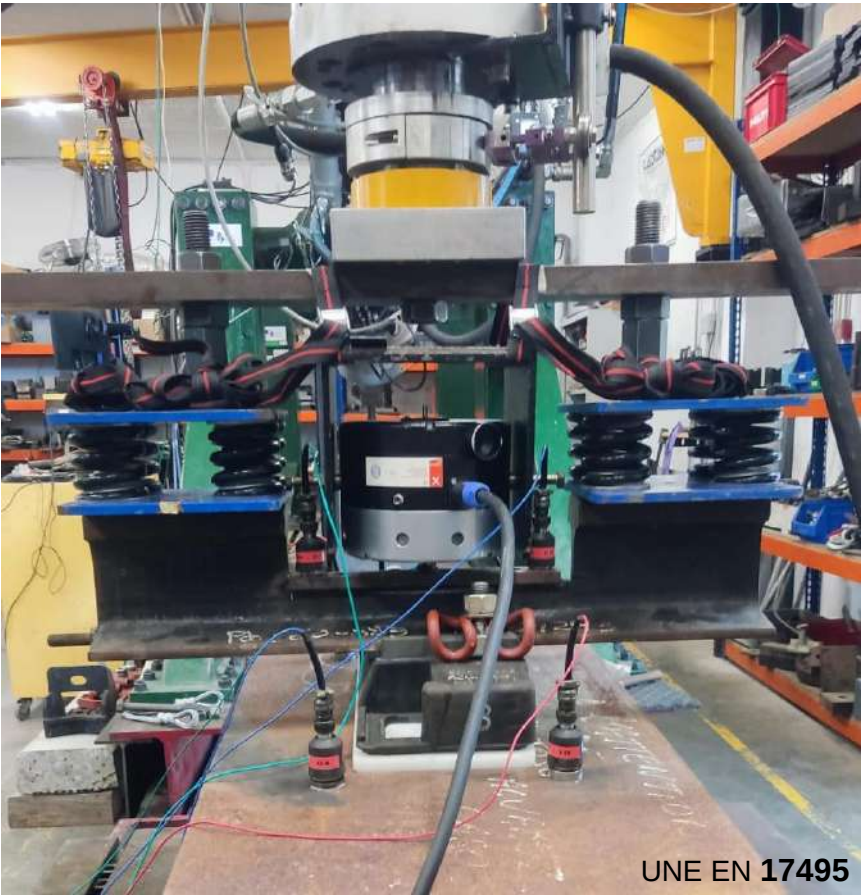


k7

NOISE + VIBRATION



NOISE + VIBRATION

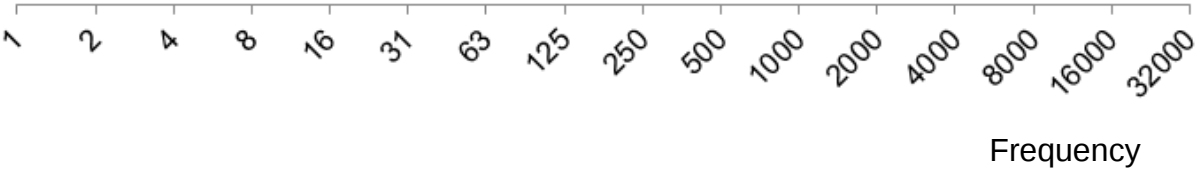


Ground-borne vibration
(1-100 Hz)

Ground-borne
noise (20-250 Hz)

Airborne noise
(20-20.000 Hz)

Case	Component	Preload values kN	Minimum frequency range (one-third octave bands) Hz	Preferred frequency range (one-third octave bands) Hz
Rolling noise / train interior rolling noise (1)	Rail pad	18	80 - 400	50 - 1 250
	Fastening assembly or baseplate pad	5		
Bridge noise (2)	Rail pad	18; 35; 50; 68	80 - 400	25 - 2 000
	Fastening assembly or baseplate pad	5; 18; 35; 50		
Ground borne noise (3)	Rail pad	18; 35; 50; 68	50 - 200	10 - 400
	Fastening assembly or baseplate pad	5; 18; 35; 50		



01

Low stiffness range

02

Acoustic-vibration response

03

Corrugation prevention

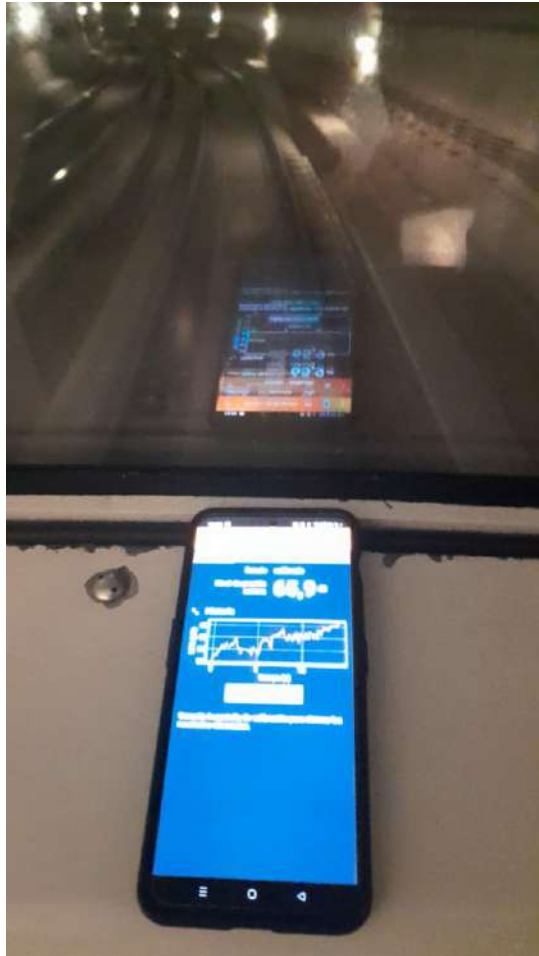
THERE ARE MULTIPLE FACTORS THAT INFLUENCE WAVE WEAR

At the level of the track superstructure, it is confirmed that increase the elasticity of the track, especially **at the transverse level, helps to delay wave wear.**

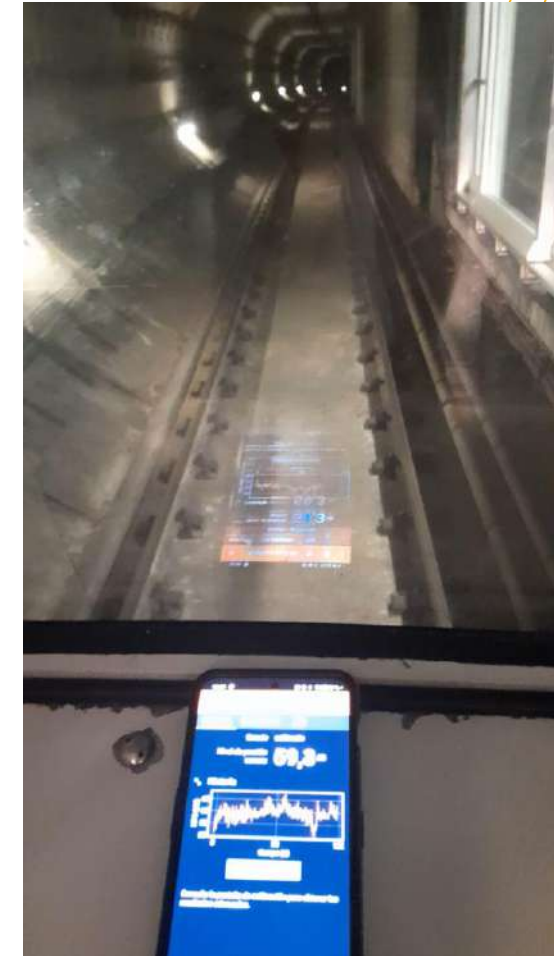
It is important to not exceeding the maximum dynamic over-width safety limits and understand that not always this formula Works as there are many factors that influence on this kind of defect.



ROLLING NOISE EFFECT OF LATERAL STIFFNESS



ZONE A LOW SYSTEM FST



ZONE B: BONDED PLATES



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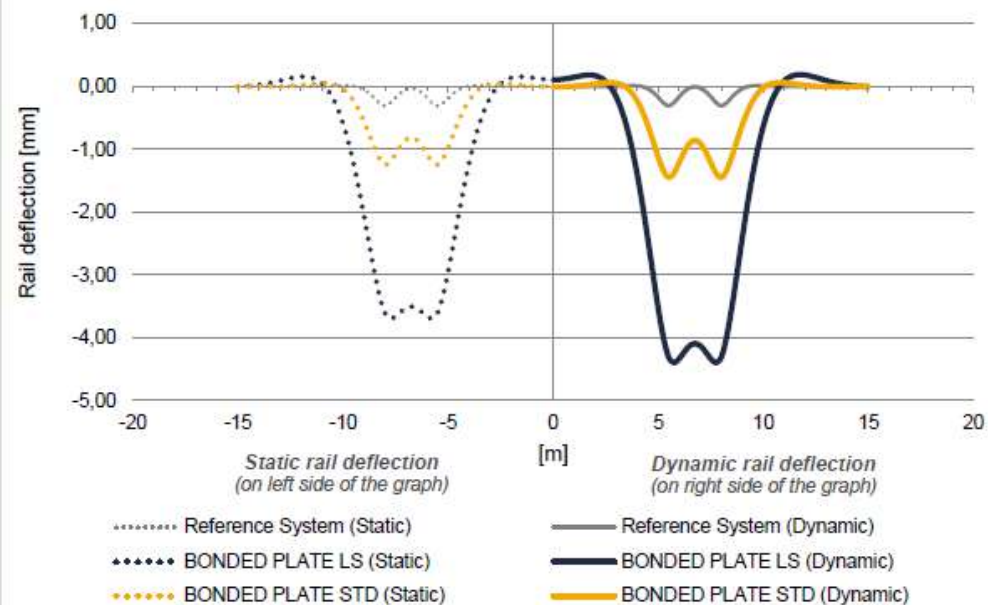
euskal trenbide sarea

General Project information

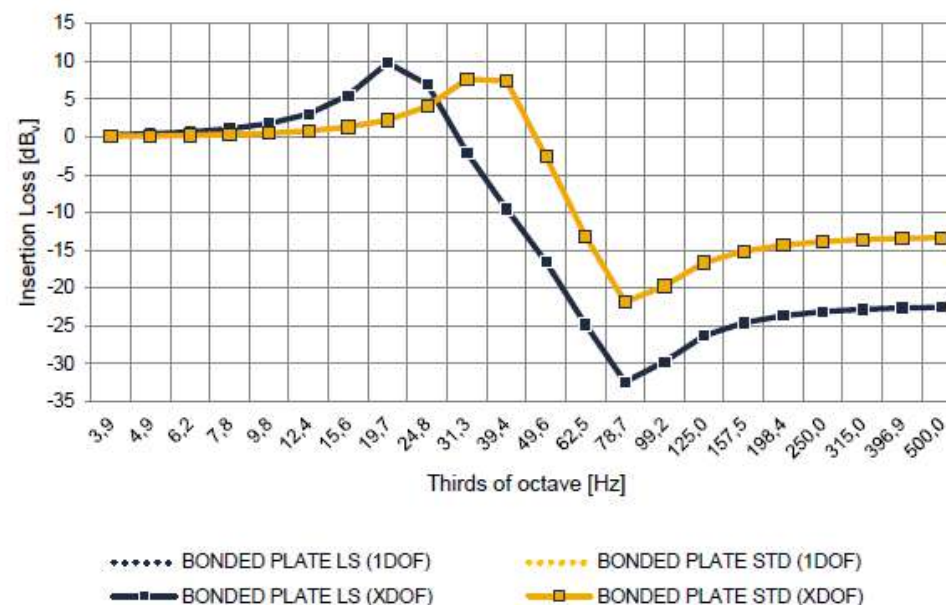
Project Number	Country	Project Name	Customer	Customer
Author	Date	Calculation Case	Calculation Case	Program Version 5.2.2

Outputs

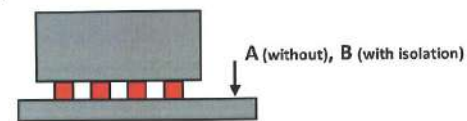
Rail deflection



Insertion Loss



IL



Eisenmann parameters for dynamic deflection calculations

Standard deviation, t	3,0	3,0	3,0
Track condition, ϕ	10 %	10 %	10 %
Dyn. Amplification Factor	1,34	1,34	1,34

Equivalent track stiffness (track deflection by one single axle)

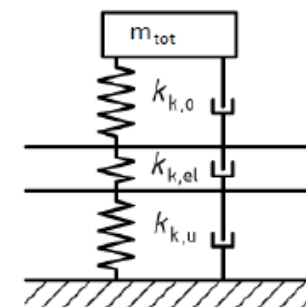
Static	520,89 kN/mm	43,37 kN/mm	128,63 kN/mm
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Total deflection at rail head

Static	0,31 mm	3,69 mm	1,24 mm
Dynamic	0,31 mm	4,39 mm	1,45 mm

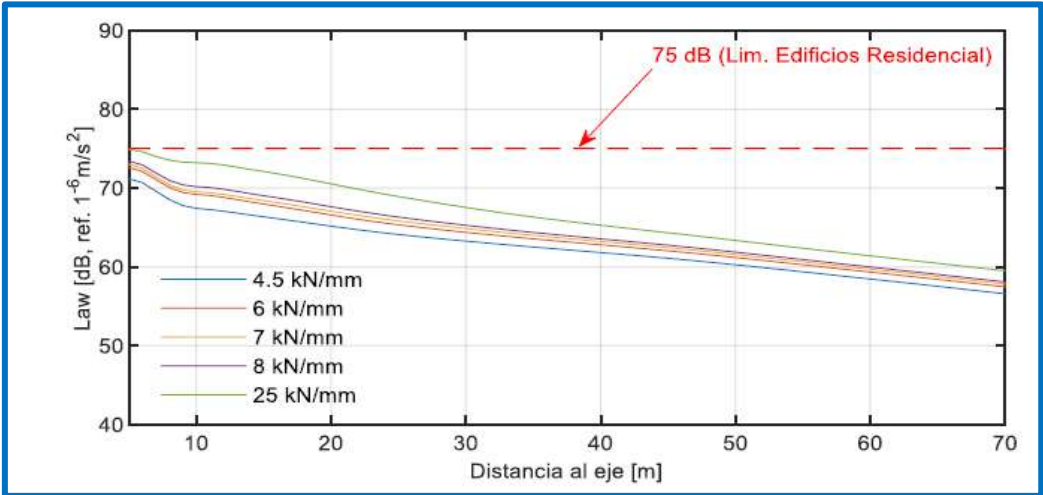
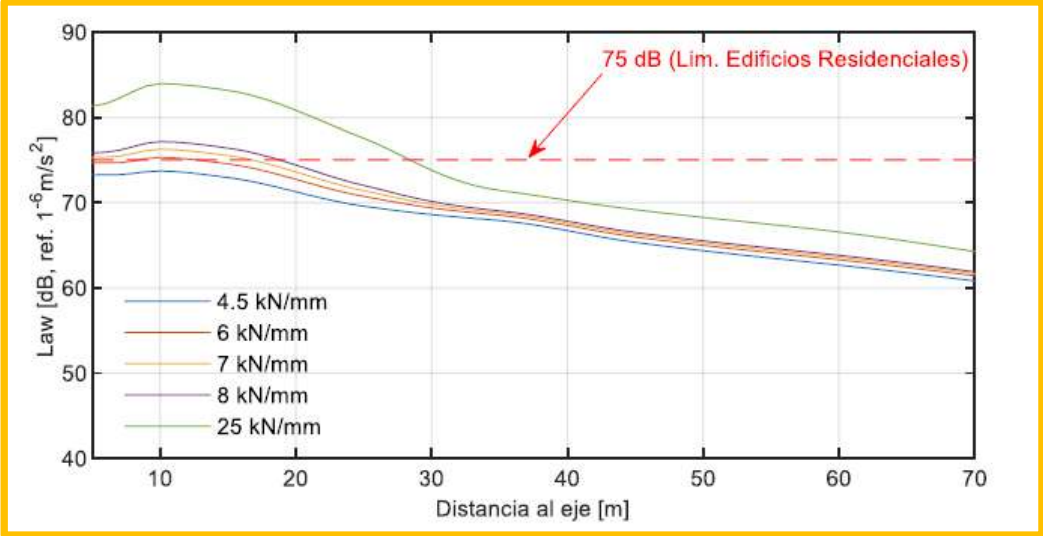
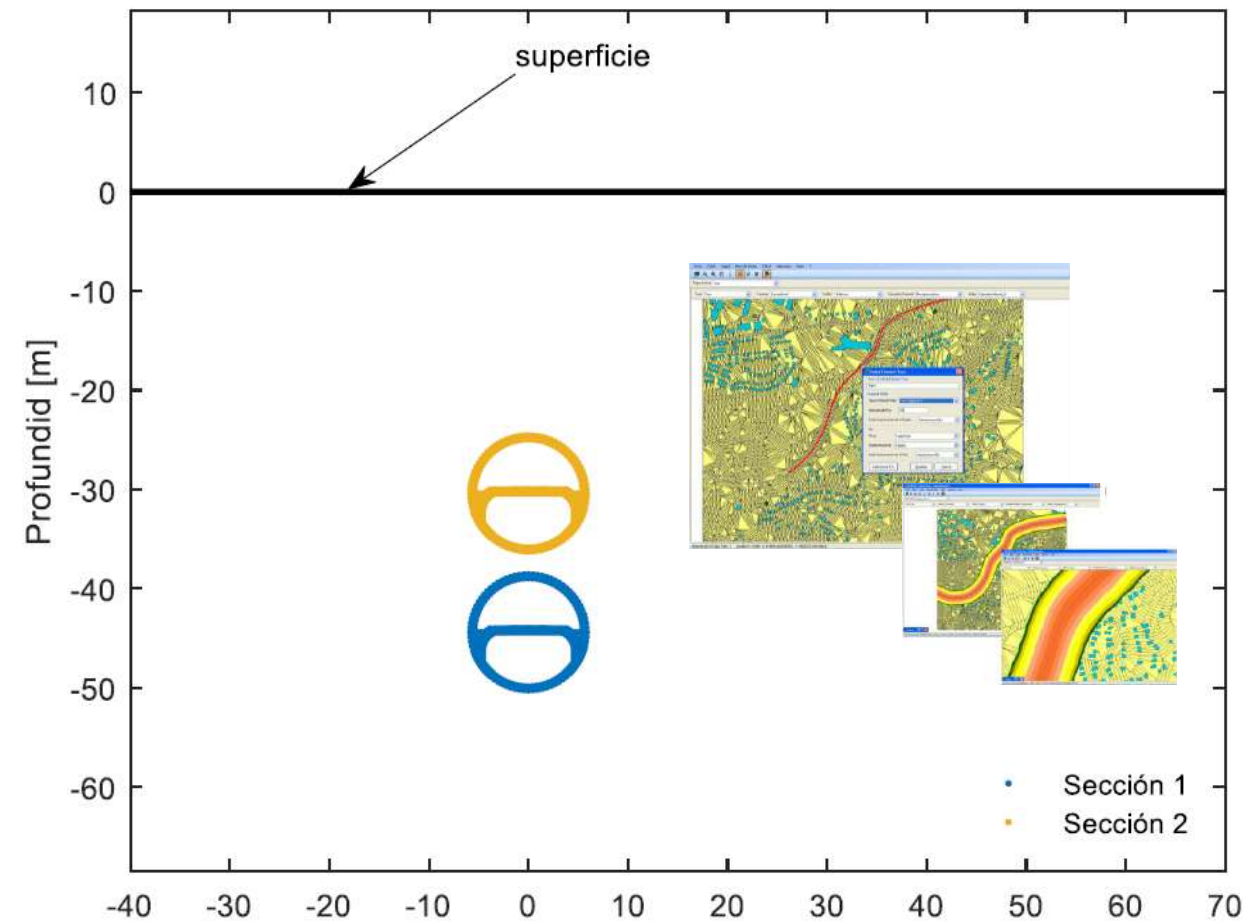
Results vibration

1DOF	XDOF	1DOF	XDOF
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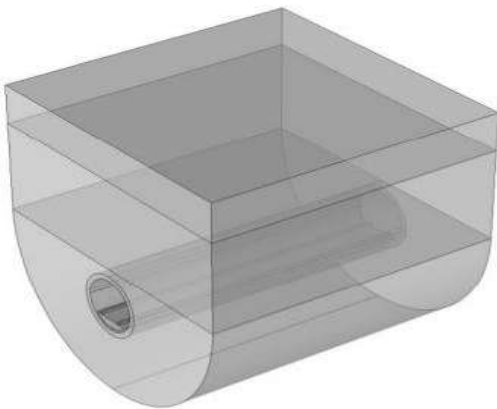
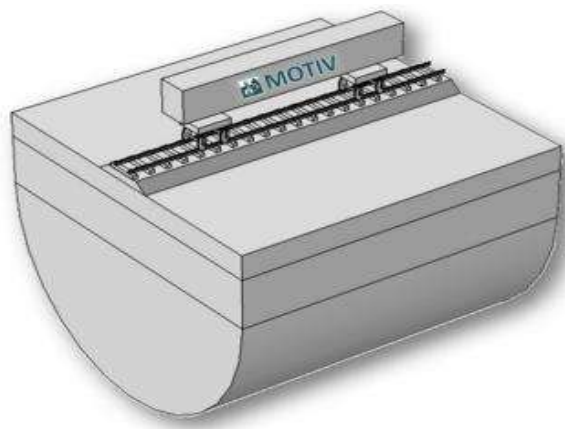
Rail Track Vibration Mitigation

Limit cases

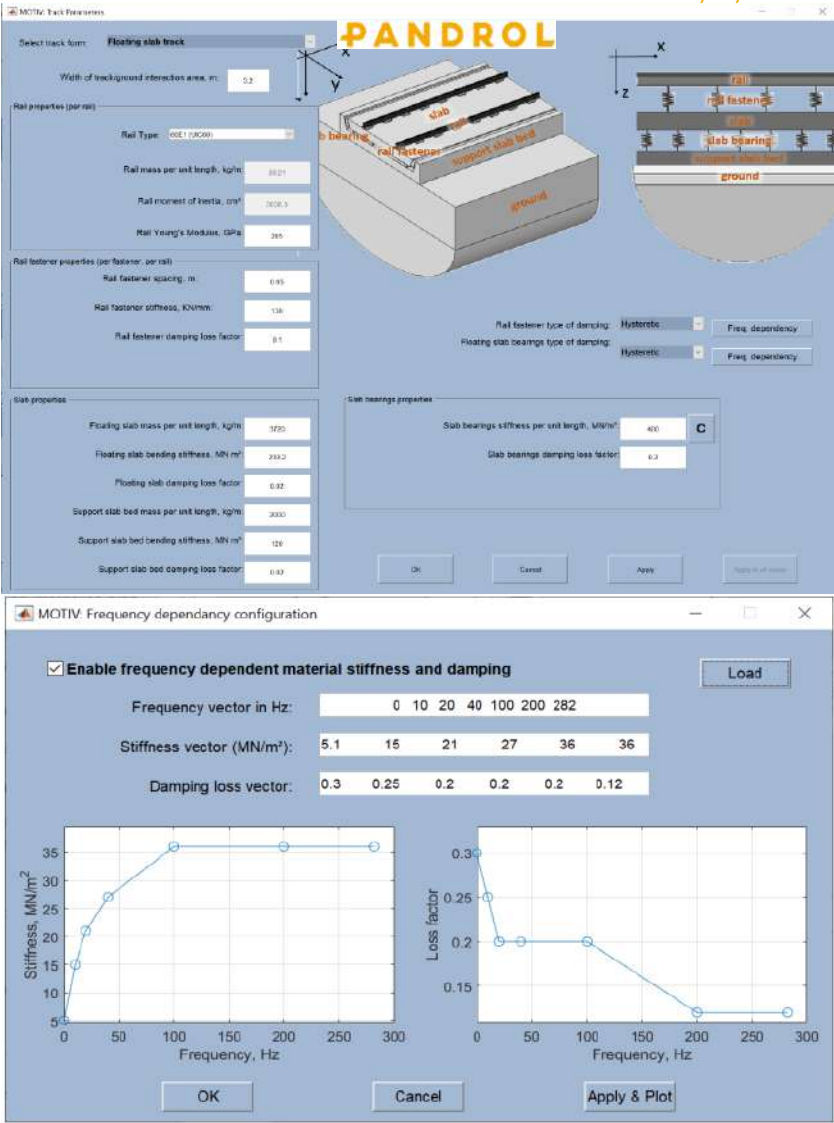


Rail Track Vibration Mitigation

PANDROL MOTIV (Modelling Of Train Induced Vibration)



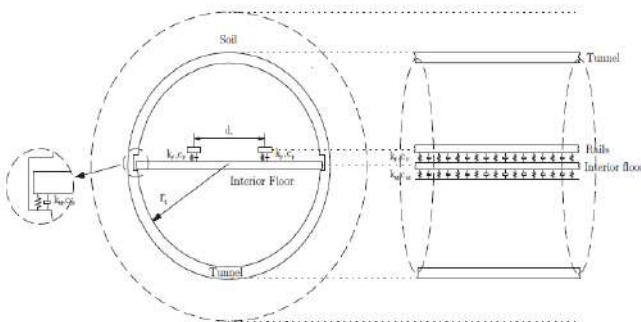
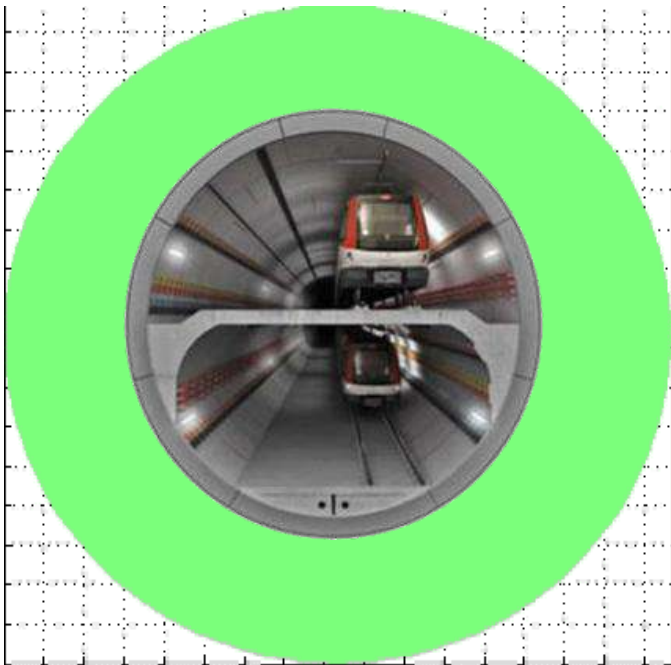
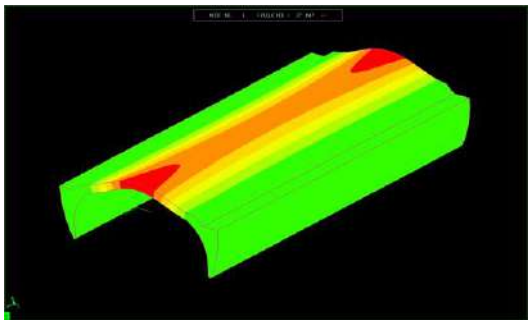
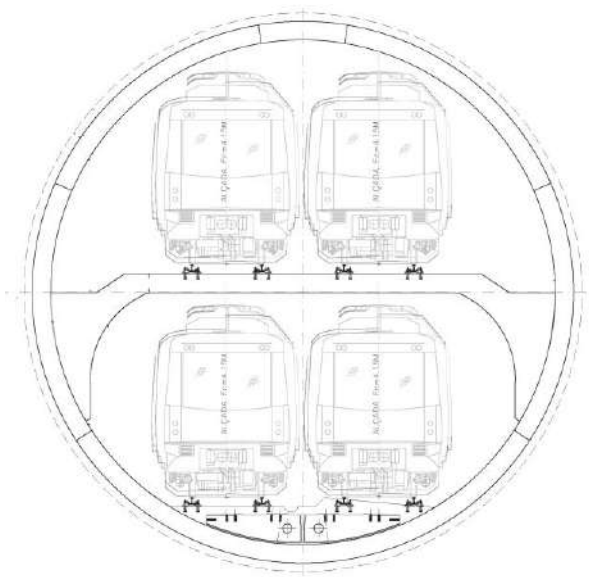
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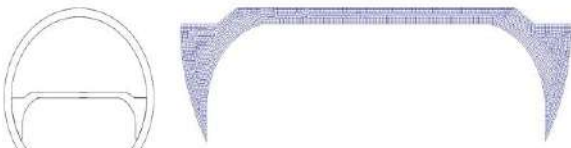
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OPTIMAL STIFFNESS. RESULTS

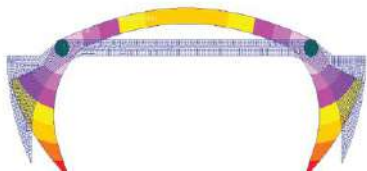
Control of vibration induced by railway traffic in the upper floor.



Transversal eigenfrequencies of the plate found using a FEM model.

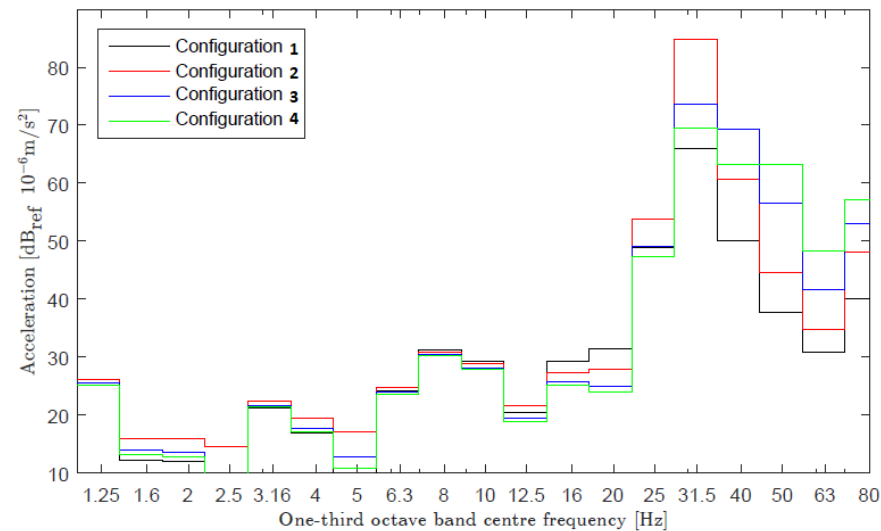


Shape of the first eigenmode



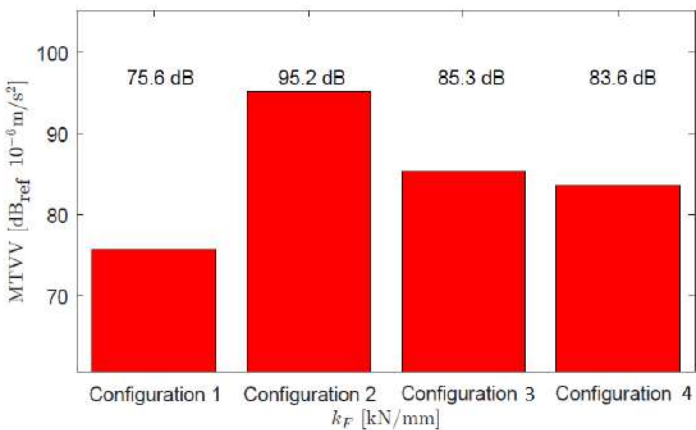
OPTIMAL STIFFNESS. RESULTS

Pondered ISO-2631-2 one-third octave band spectrums

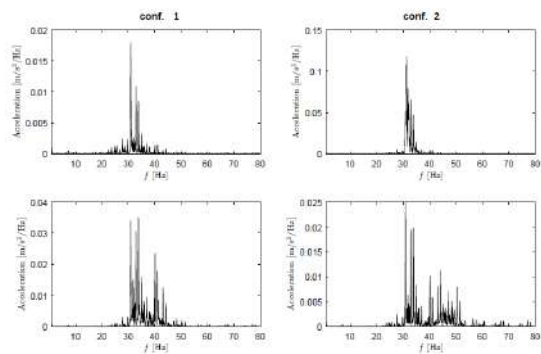


The Maximum Transient Vibration Value is obtained for a set of points (ISO-2631-1)

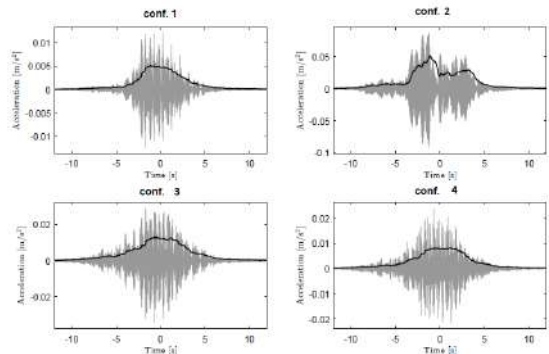
$$MTVV = \max \left[\sqrt{a_X^2 + a_\theta^2 + a_R^2} \right]$$



Response in frequency



Response in time



K4,5



K10



K17



K25

Kd:(KN/mm)

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THANK YOU

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